

**AMENDMENTS**Amendments to the Specification

- I. Please insert the following paragraphs on page 1, immediately before the paragraph beginning at indicated line 1:

**BACKGROUND OF THE INVENTION**

- II. Please insert the following paragraph on page 3, at indicated line 8:

**BRIEF SUMMARY OF THE INVENTION**

- III. Please insert the following paragraph on page 6, immediately before the paragraph beginning at indicated line 1:

**BRIEF DESCRIPTION OF THE DRAWINGS**

- IV. Please insert the following paragraph on page 7, at indicated line 26:

**DETAILED DESCRIPTION OF THE INVENTION**

- V. Please amend the paragraph on page 8, at indicated lines 24 – 32 as follows:

Referring now to FIG. 3 there is illustrated one such drive system including a scavenging oil circuit. The main driving source is the hydraulic motor 22 21 which powers the rotating system as a whole, including the scroll drive pump 23 (not shown). At the suction end of the pump 23 41 the scavenging oil is delivered and directed through the filter 24 around the cooler 25 and then via the biasing valve (not shown) 28 into the tank tank 27. The scavenging pump 26 (not shown in FIG. 3, but see FIG. 12) suctions fresh conditioned oil from the tank 27 which is fed in

the return flow from the hydraulic motor 21. The leakage conduit (not shown, but see FIG. 12 for an example of a leakage conduit) to the tank (broken line) runs separate from the scavenging circuit.

VI. Please amend the paragraph on page 9, at indicated lines 1 – 12 as follows:

Referring still to FIG. 3 it is thus illustrated symbolically how not only the feed pump for the hydraulic motor is assigned thereto--preferably in a common casing--for rotation, but also open/closed loop control of the liquid flow delivered by the feed pump to the hydraulic motor so that the hydraulic motor can be operated with adjustable variable speed and thus, by way of example, the decanter can be operated with adjustable variable differential speed between bowl and scroll. This is symbolically expressed in FIG. 3 by the controllable aperture 40 indicated between the hydraulic motor 21 and the corotating feed pump 41. Via the rotary feedthrough 44 the working fluid is basically drawn off from the tank 27 via ~~the cooling and/or filter means filter 24, and/or cooler 25~~ and returned to the tank or vice-versa.

VII. Please amend the paragraph on page 9, at indicated lines 14 – 19 as follows:

Referring now to FIG. 4 there is illustrated the embodiment in more detail of a controlled flow-control valve 42 in combination with an assigned aperture 40 which branches off the hydraulic fluid not required by the hydraulic motor 21 into the non-pressurized area (2-way flow control) whilst FIG. 5 shows the arrangement of a closed loop controlled aperture 40 in the feed circuit between pump 41 and hydraulic motor 21 (3-way flow control).

VIII. Please amend the paragraph on page 9, at indicated lines 27 – 31 as follows:

Referring now to FIGS. 9 and 10 there is illustrated how a proportional magnet or solenoid 36 is used to control (via direct actuator 45 in FIG. 9) the corotating valve system 43 with the aperture 40 and respectively how an actuator 37 controls an aperture 40 37 is controlled through which the controlled flow flows, with the aid of the return pressure as dictated by the valve 46 exterior to the rotating system.

- IX. Please replace the paragraph on page 10, at indicated lines 1 – 12 with the following paragraph:

It is to be noted that as of FIG. 5 in FIGS. 11 – 16, examples of embodiments are shown in which the feed pump comprises a variable displacement volume, two opposed cylinders (actuators) 29 and 30 being provided for this purpose. The small caliber cylinder (actuator) 30 receives the operating pressure of the feed pump 23 and is spring-assisted. The larger caliber cylinder (actuator) 29 is activated via a servo slide gate or valve 31, at the right-hand side of which a spring may be active whose compressive force increases to the left with increasing effect of the cylinder (actuator) 29 (travel return). Pressurizing the servo slide valve 31 displaces the cylinder (actuator) 29 until the force at the servo slide valve 31 is equalized (counter-pressure spring FIG. 13). The displacement of the cylinder (actuator) thus corresponds to the pressure applied to the servo slide valve 31.

- X. Please insert the following paragraph on page 10, at indicated line 13:

FIGS. 12 – 14 illustrate various embodiments of the following features: at the suction end of the pump 23 the scavenging oil is delivered and directed through the filter 24 around the cooler 25 and then via the biasing valve 28 into the tank 27. The scavenging

pump 26 suctions fresh conditioned oil from the tank 27 which is fed in the return flow from the hydraulic motor 21. The leakage conduit 50 to the tank (broken line) runs separate from the scavenging circuit.

XI. Please revise the following paragraph on page 10, starting at line 25:

Referring now to FIG. 16 there is illustrated a further example embodiment in which also the charging pump 26 is included in the rotating system of hydraulic motor 21, and feed pump 23 and 41 respectively with open/closed loop control of the feed flow.